Effects of Cooling Fluid Flow Rate on the Critical Heat Flux and Flow Stability in the Plate Fuel Type 2 MW TRIGA Reactor

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ABSTRACT
The conversion program of the 2 MW TRIGA reactor in Bandung consisted of the replacement of cylindrical fuel (produced by General Atomic) with plate fuel (produced by BATAN). The replacement led into the change of core cooling process from upward natural convection type to downward forced convection type, and resulted in different thermohydraulic safety criteria, such as critical heat flux (CHF) limit, boiling limit, and cooling fluid flow stability. In this paper, a thermohydraulic safety analysis of the converted TRIGA reactor is presented by considering the Dynamic Nucleate Boiling Ratio (DNBR) criterion, Onset Nucleate Boiling Ratio (ONBR) limit, and cooling fluid flow stability at various cooling fluid flow rate. The numerical analyses were performed using the HEATHYD program on the hottest channels of reactor core. The combination of heat transfer and fluid flow analysis were conducted for reactor operation at 2 MW with 20 fuel element bundles and four control rod bundles. Incoming fluid flow to the cooling channel was fixed at 44.5 °C temperature and 1.9970 bar pressure, and its flow rate was varied from 1.25 to 3.5 m³/h. By inputting these values, as well as the total power of fuel elements per bundle, the wall temperature distribution of the plate fuel element, cooling fluid temperature distribution, and pressure losses in the channels were obtained for the analysis of CHF limit, boiling limit, and flow stability. It was shown that no boiling occurred for the cooling fluid flow rate range of 2.4 to 3.5 m³/h, and even at the cooling fluid flow rate of 1.25 m³/h where some bubbles occurred, the DNBR was higher than the critical limit (more than 23) while the flow stability criterion in some channels were slightly less than 1 (unstable). At the cooling fluid flow rate of 1.4 m³/h, however, the flow became stable in all channel. The results showed that even though some bubbles start to occur, the plate-fuel type 2 MW TRIGA reactor can safely operate in the terms of CHF limit and flow stability.

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INTRODUCTION
Bandung TRIGA 2000 reactor is a research reactor built in 1964 with current maximum power of 2000 kW, and is considered an important installation for nuclear research and education in Indonesia, due to its capability to back up the production of radioisotopes. Its operation, however, mostly depends on the fuel supplied by General Atomics (GA). Currently GA is no longer producing the TRIGA reactor fuel, but at the same time BATAN Serpong is already able to produce its own plate-type fuel. This capability should be beneficial for BATAN Bandung for keeping the TRIGA 2000 reactor operational and for performing research for long period; the refore, several researchers at BATAN Bandung planned a conversion program by converting the fuel from cylindrical to plate type, releasing it self from dependency on fuel supplies from other countries.
The conversion program should alter the state of neutronic and thermohydraulic equilibrium in the core. Neutronically, the core structure will change the power distribution, while the cooling process will also be changed. These conditions will affect the safety limit of the core, especially its thermohydraulic aspects such as boiling limit, critical heat flux (CHF) limit, and cooling flow stability [1-4]. The Bandung TRIGA 2000 reactor used a natural convection cooling system type, but with the conversion into a square core, the cooling will be performed by means of forced convection, and the cooling fluid will flow in a very narrow channel of about 2.3 mm between the fuel plates.

To predict the safety limits, a cooling system analysis of the converted TRIGA 2000 reactor is required. HEATHYD program for reactor core with square fuel element, forced convection, and steady state flow was used to simulate the cooling system characteristics. Analyses were performed by inputting the power distribution of the reactor core calculated in neutronic analysis, and the necessary cooling fluid flow to avoid the heat accumulation can be calculated so that the limit of safety operations will not be exceeded.

**EXPERIMENTAL METHODS**

**Materials**

The original fuel cylinders arrangement of the Bandung TRIGA reactor inside the core is shown in Fig. 1. About 117 fuel elements were arranged in a hexagonal form, along with five control rods. The reactor is cooled by water through the primary and secondary cooling systems as shown in Fig. 2.

![Fig. 1](image1.png)  
(a) Core configuration of Bandung TRIGA Reactor, (b) Cylindrical fuel element.

![Fig. 2](image2.png)  
Fig. 2. Bandung TRIGA Reactor cooling system.

The hexagonal arrangement of the Bandung TRIGA reactor core was converted into a square core arrangement as shown in Fig. 3. The cooling system, which initially used natural convection with upward flow direction, was converted into a forced convection with downward flow direction as shown in Fig. 4 [5].

![Fig. 3](image3.png)  
(a) Core configuration of converted TRIGA Reactor, (b) Plate-type fuel.

![Fig. 4](image4.png)  
Fig. 4. Design of converted TRIGA Reactor cooling system.
Figure 4 shows the cooling flow of the converted TRIGA reactor which differs from original Bandung TRIGA reactor by using a primary pump (P1) drawing the cooling fluid downward. This indicates that the converted TRIGA reactor uses forced convection cooling system, while the original Bandung TRIGA reactor used natural convection cooling system.

Methodology

Due to the change of behavior and direction of the cooling system, it is necessary to analyze the safety and performance of the converted cooling system. The analyses were performed numerically using the HEATHYD program [6], in the following manner (i) Collecting data such as shape, dimension, material, and thermohydraulic characteristic of the Bandung TRIGA 2000 reactor and plate-type fuel data that will be used; (ii) Providing model of the channel with highest temperature in the Bandung TRIGA 2000 reactor core using plate type fuel; (iii) Specifying boundary conditions based on the assumptions that have been stipulated; (iv) Performing thermohydraulic analysis by using HEATHYD program to see the core cooling performance in the channel with highest temperature; (v) Providing technical data for the converted TRIGA reactor that will be used in the design of the reactor cooling system.

HEATHYD is a software package developed for analysis of reactor core with square fuel type and convection cooling system in steady state flow, utilizing a combination of heat transfer and fluid flow calculations. The detailed flow diagram of the HEATHYD program is shown in Fig. 5. The inputs for HEATHYD program include (i) Number of channels in one fuel element (20), number of inner iterations (100), number of outer iteration (100), number of outputs after the outer iteration (12), and flow direction (up-down); (ii) Flow rate of the cooling (1.25-1.6 m³/h), plate width (67.1 mm), plate thickness (1.3 mm), length of the channel (694 mm), inlet temperature (44.5 °C) and pressure (2 bar); (iii) Distance between the plates fuel elements (2.55 mm); (iv) Thickness of plate fuel element (1.3 mm); (v) Number of mesh points along the axial direction (60), parameter fuel element in axial direction (cos or sin), the critical heat flux (Mirshak/Labuntsov correlation); (vi) Polynomial factor [6]; (vii) Power of fuel element (0.125 MW), meat length (620 mm), length of cladding (694 mm); (viii) Thermal conductivity (0.693 W/m.K), and density of cooling fluid (978 kg/m³); (ix) Meat width (62.75 mm), width of fuel element plate (70.75 mm); (x) Radial flux, result of neutronic calculation [12].

The results of the analysis were then used to design the converted TRIGA Reactor core cooling system. To conservatively determine the safety limit, the analysis was performed only at the channel with the highest temperature.

In the fluid flow section, flow velocity calculation at any point was conducted with constant mass flow rate assumption (ρAv = constant). Related pressure drops in the channel were calculated based on [7-8].

Friction, by using equation Blassius/Colebrook;

\[
\Delta P_f = 4 f \left(\frac{L_w}{D_w}\right) \left(\frac{1}{2} \rho v^2\right) x 10^{-5}
\]  

where \( f = 0.316 \) Re\(^{-0.25}\), \( L_w = \) heated length of the channel (mm), \( D_w = \) heated equivalent diameter of the channel (mm), \( \rho = \) density of fluid (kg/m³), \( v = \) velocity of fluid (m/sec), and \( \text{Re} = \) Reynold number (dimensionless).

Velocity change obtained from the following Bernoulli equation,

\[
\frac{1}{2} \rho v^2 + P = \text{constant}
\]  

with \( P = \) pressure at channel.

Inlet and outlet channel cross section change, related by,

\[
\Delta P_m = k_m \left(\frac{1}{2} \rho v^2\right) x 10^{-5}
\]

where

\[
k_m = \left(\frac{1}{\beta} - 1\right)^2 + 0.05
\]

and

\[
\beta = 0.63 + 0.37 \left(\frac{A_c}{A_0}\right)^2
\]

where \( A_c = \) cross-sectional area in an element (m²), \( A_0 = \) cross-sectional area of the end box immediately beyond the channel entrance/exit (m²).
Gravitational potential energy change, as a function of height,

$$\Delta P_h = \Delta \rho gh$$

with \( g \) = gravitational acceleration (m/s²), \( h \) = height of channel (m).

In the heat transfer section, items to be calculated are as follows [9-11].

Determination Prandtl number (Pr) by,

$$Pr = \frac{C_p \mu}{k}$$

(4)

where \( C_p \) = specific heat of fluid (kJ/kg °C), \( \mu \) = dynamic viscosity of fluid (kg/m sec), \( K \) = heat conductivity of fluid (W/m °C).

Convection heat transfer coefficient (h) calculated by Dittus-Boelter equation

$$h = 0.023 \text{Re}^{0.8} \text{Pr}^{0.4}$$

(5)

Heat flow rate (\( q'' \)) at any point calculated by

$$q''(z) = q''_{max} \cos \frac{\pi z}{L}$$

and

$$q''_{max} = E \Sigma \phi_{max}$$

(6)

where \( E \) = energy per fission (200 MeV), \( \Sigma \) = macroscopic crossection (1/cm), \( \phi \) = neutron flux (n/cm² sec); the fuel cladding temperature (\( T_p \)) and cooling fluid temperature (\( T_s \)) at point \( z \), is defined as

$$T_{f0} = \frac{q(z)}{mC_p} + T_{b} \text{ dan } T_{p0} = \frac{q(z)}{hA} + T_{f0}$$

(7)

where \( m \) = mass flow rate (kg/s), \( T_b \) = in fluid temperature (°C), \( T_{f0} \) = out fluid temperature (°C), \( T_{p0} \) = out plate temperature (°C).

Temperature difference between cladding (\( T_p \)) and saturated temperature cooling fluid (\( T_s \)) calculated by Jens-Lottes equation as follows,

$$T_p - T_s = 25 \exp\left(\frac{-P}{62}\right)\left(\frac{q''}{100}\right)^{1.4}$$

(8)

Heat flux distribution (\( q_{ONB} \)) in the two phase condition formulated by Rohsenow [6] equation as follows,

$$q_{ONB} = \frac{1.156}{9.234} \left[1.8\left(T_p - T_s\right)\right]\left(0.463P_f\left(0.0234\right)^{-1}\right)$$

(9)

Critical heat flux (\( q_{DNB} \)) is calculated by Mirshak equation as follows,

$$q_{DNB} = 151(1+0,1198\nu)(1+0,00914\Delta T_{sub})(1+0,186P)$$

(10)

with \( \Delta T_{sub} = T_p - T_{b} - T_{f0} \)

Fluid flow stabilization (\( S_f \)) is formulated by Whittle and Forgan [6] equation as follows,

$$S_f = R (T_p - T_{b})(T_{f0} - T_{b})$$

with

$$R = (1 + \frac{25D_{in}}{L_w})^{-1}$$

(11)

Boiling condition will not occur if;

$$\frac{q_{ONB}}{q_{actual}} > 1$$

(12)

Burnout condition will not occur if;

$$\frac{q_{DNB}}{q_{actual}} > 1$$

(13)

Fluid flow will be in unstable condition if,

$$S_f < 1$$

(14)

**RESULTS AND DISCUSSION**

The results of the analysis with HEATHYD program include the magnitude of the heat flux during one phase and two phase wall temperature fuel elements and the temperature of the cooling fluid in maximum condition, as shown in Table 1. This analysis was conducted for the reactor operating at 2 MW with 20 bundles of fuel elements including four pieces of control rod bundles, with the cooling fluid flow rate entering the channel varied from 1.25 to 3.5 m³/h, inlet temperature of 44.5 °C, and pressure of 1.9970 bar. The magnitude of the heat flux boiling (\( q_{ONB} \)) was calculated by using Rohsenow equation (9), the critical heat flux (\( q_{DNB} \)) calculated by using Mirshak equation (10) and the flow stability (\( S_f \)) calculated by using Whittle and Forgan equation (11).
Table 1. Heat Flux, Plate Temperature and Coolant Temperature

<table>
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<tr>
<th>Position</th>
<th>$q_s$ (Watt/cm$^2$)</th>
<th>$q_{sa}$ (Watt/cm$^2$)</th>
<th>$T_f$(left) (°C)</th>
<th>$T_p$ (°C)</th>
<th>$T_f$(right) (°C)</th>
<th>$q_i$ (Watt/cm$^2$)</th>
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</table>

Fig. 6. Boiling in the core of a bundle channel fuel element for cooling flow variations.

To identify the boiling process, an Onset Nucleate Boiling Ratio (ONBR) was defined as the ratio of boiling heat flux ($q_{ONB}$) to actual heat flux. When ONBR < 1, actual heat flux exceeds the boiling heat flux, thus allowing the occurrence of boiling. Figure 6 shows that at flow rates from 1.25 to 1.6 m$^3$/h, the ONBRs were mostly lower than one and boiling should occur.

It can be seen from the figure that to prevent boiling, the cooling fluid flow rate should be increased. By increasing the flow rate up to 3.5 m$^3$/h, apparently the ONBR was increased, exceeding 1 at the cooling fluid flow rate of 2.4 m$^3$/h as shown in Fig. 7.

A further burnout condition, when the cooling fluid is not in contact with the wall and a rapid increment of cladding temperature results for even a small increment of heat flux, should be anticipated when the boiling condition has been fulfilled. The burnout fraction is defined as Dynamic Nucleate Boiling Ratio (DNBR), i.e., the ratio between critical heat flux (CHF) and actual heat flux. When DNBR < 1, actual heat flux exceeds the CHF, thus allowing the occurrence of burnout.

Fig. 7. Boiling in the core of a bundle channel fuel element for higher cooling flow variations.

Figure 8 shows that for flow rates of 1.25 m$^3$/h to 1.6 m$^3$/h, DNBR exceeded 23, much
higher than 1. It can be concluded that no burnout should be anticipated during the 2-MW operation with higher cooling fluid flow rate, and rapid temperature increasing in the plate fuel and core material are not likely to occur.

As for the flow stability shown by $S_f$ in equation (11), if $S_f$ less than 1, then flow instabilities will occur. It can be seen in Fig. 9 that the cooling fluid flow across all channels remained stable, because for all the channels in a single bundle, $S_f$ remained higher than 1.

Figure 10 shows that the temperature of the plate was always higher than the temperature of the cooling fluid, so heat flowed from the plate to the cooling fluid. As a result, the cooling fluid temperature at the channel entry became lower than at the channel outlet. Due to this condition, the most probable location of core boiling is at the outlet (bottom side). The temperature distributions at the left and right side wall have the same tendency, with the right side wall temperature being higher than that of the left side, due to the higher heat flux on the right side.


